

# Piezoelectric Generators: Modeling, Design and Applications

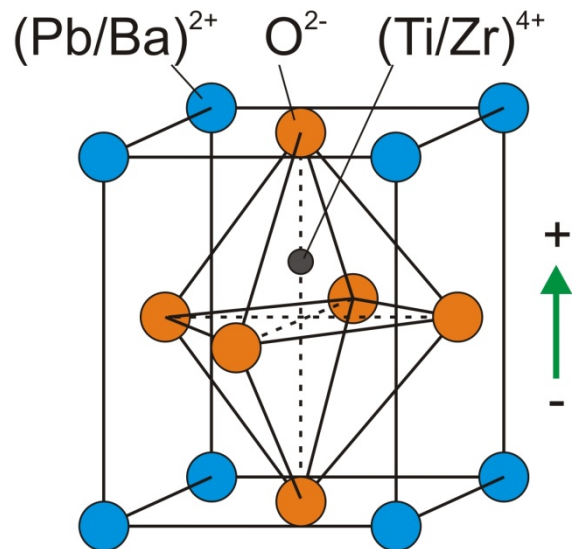
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Michael Kröner, Frank Goldschmidtboeing

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# Piezoelectric generators

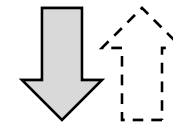
## Principle: Direct piezoelectric effect

*charge displacement in a nonsymmetric crystal lattice, obtained via a mechanical deformation of the piezoelectric material*



*Perovskite crystal structure of standard piezoceramic material (here PbZrTi or PZT and BaZrTi)*

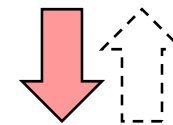
mechanical  
energy



mechanical  
deformation



charge  
displacement



electrical  
energy

# Piezoelectric energy harvesting from „mechanical energy“



## Types of „mechanical energy“

- vibration (at/from solid bodies)
- sound (transmitted through air)
- impact
- deformation
- rotation
- change of position
- ....

## Corresponding parameters

- natural frequencies
- frequency spectra or broadband
- amplitudes
- acoustic or mechanical coupling
- forces and bending moments
- rotation speed
- speed of movement
- .....

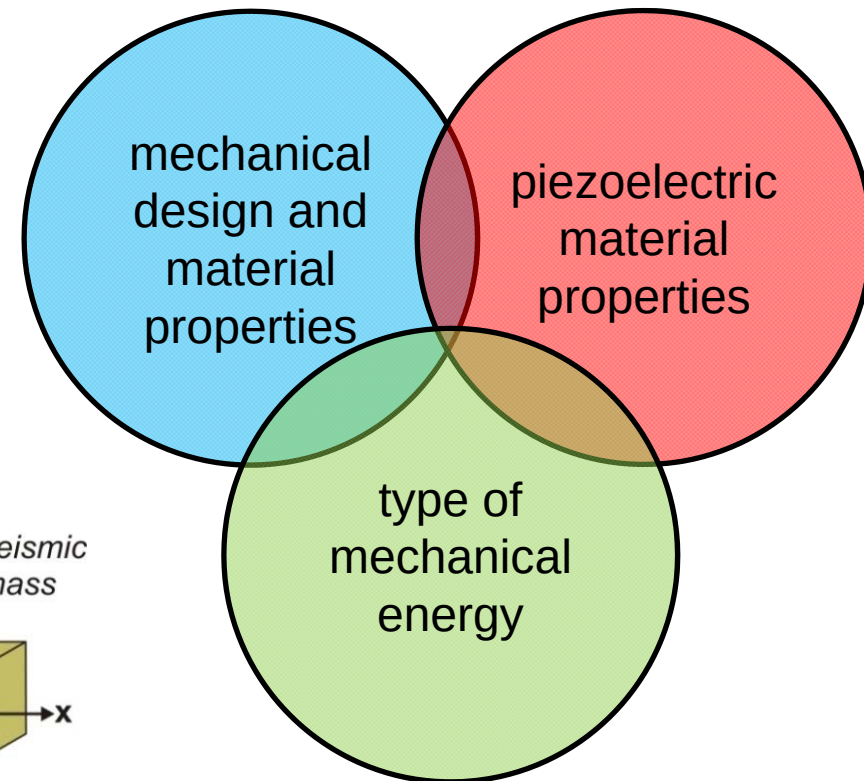
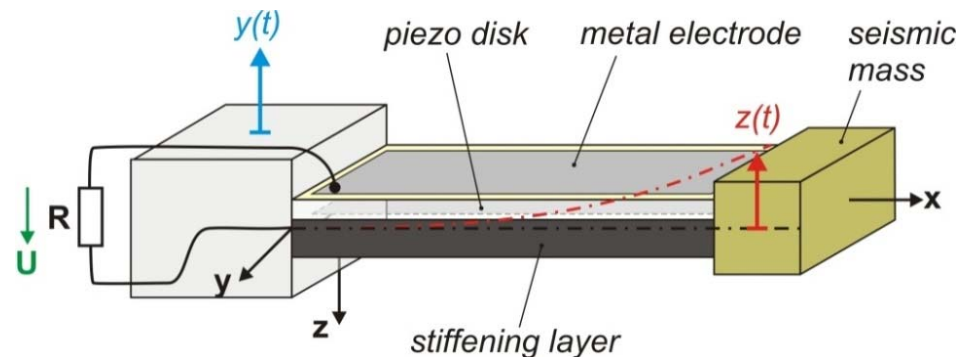
### **Resume:**

*a lot of different (design) parameters for  
a lot of different piezomaterials and  
a lot of different generator types*

# Piezoelectric bending generators as exemplary case

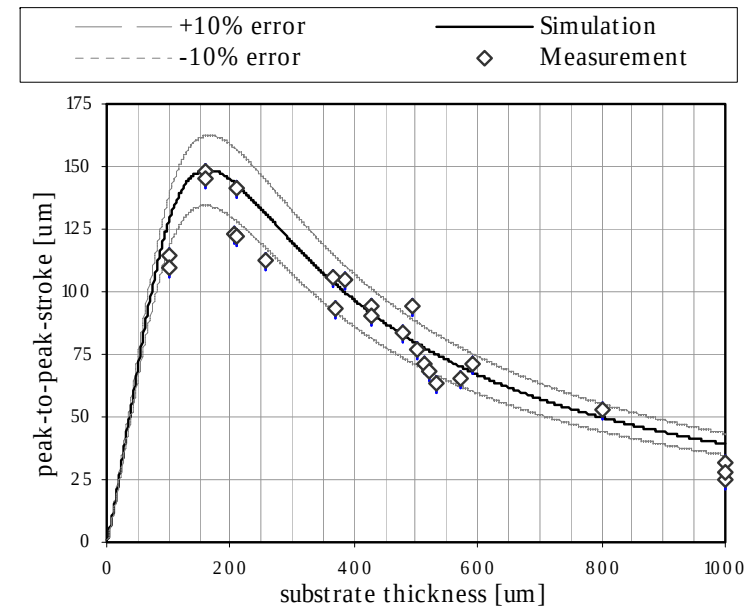
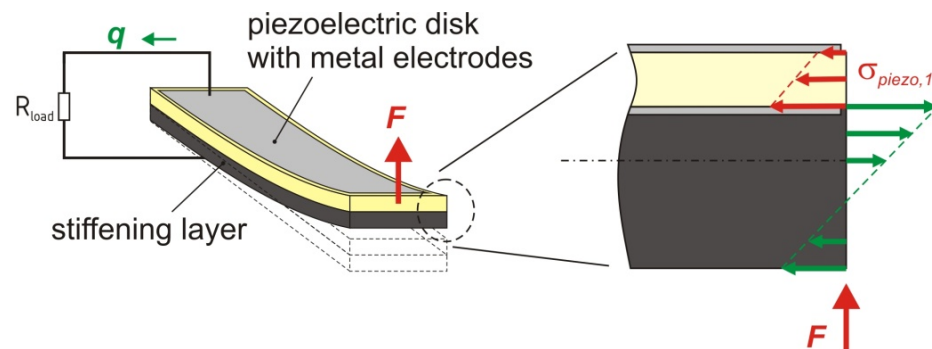
## Design

- generator as double layer or triple layer beam (1..2 piezo + 1 stiffening layer)
- „double nature“: generator is also the spring of the seismic mass of a mechanical oscillator



## Design

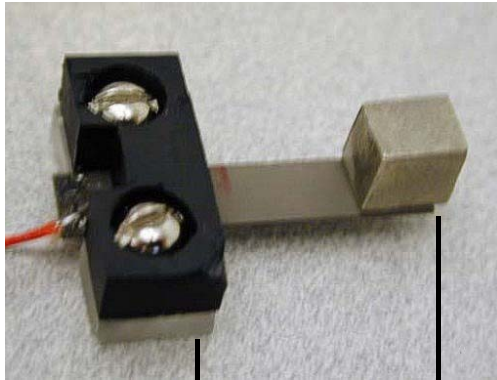
- stiffening layer required to generate unidirectional stress in the piezo layer
- optimal **height** of both layers for a fitting stress load in the piezo layer, both for generators and actuators



C. Friese, P. Woias, F. Goldschmidtboing

*IEEE Transducers '03*, Dig. Techn. Papers, Vol. 2, 2003, 1007-1010.

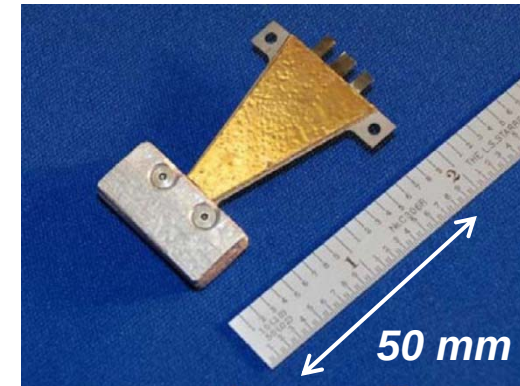
# Examples of beam-type generators



17,5 mm

$P_{\max} = 0,08 \text{ mW @ } 0,23 \text{ g and } 120 \text{ Hz}$

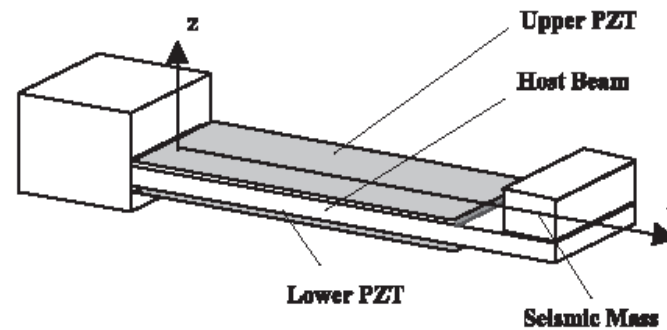
*S. Roundy et al.,  
UC Berkeley, 2004*



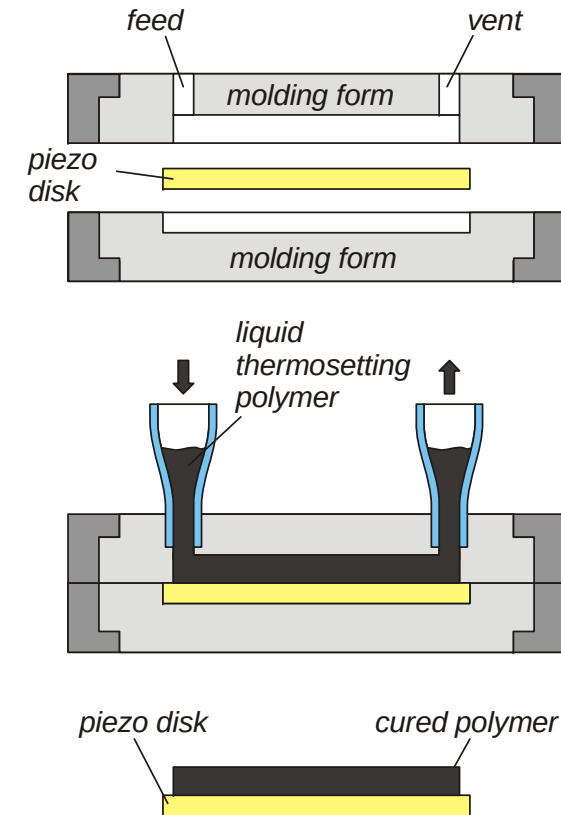
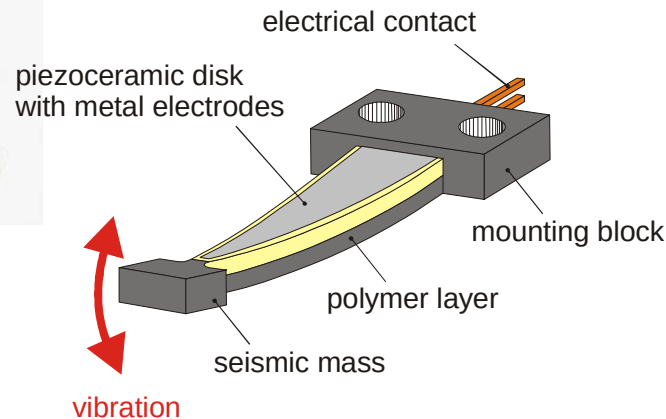
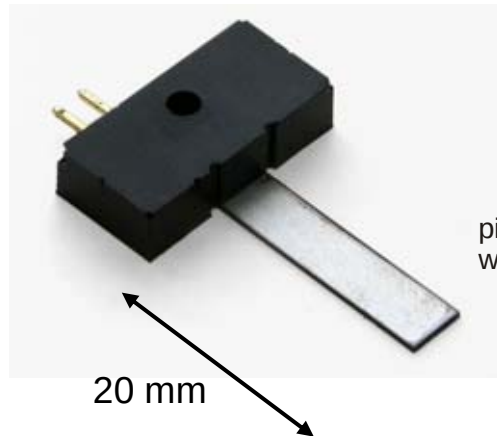
50 mm

$P = 0,45 \text{ mW @ } 1\text{g and } 60 \text{ Hz}$

„Joule Thief“  
© AdaptivEnergy, 2009



# Piezo-Polymer-Composite (PPC) technology



## Advantages

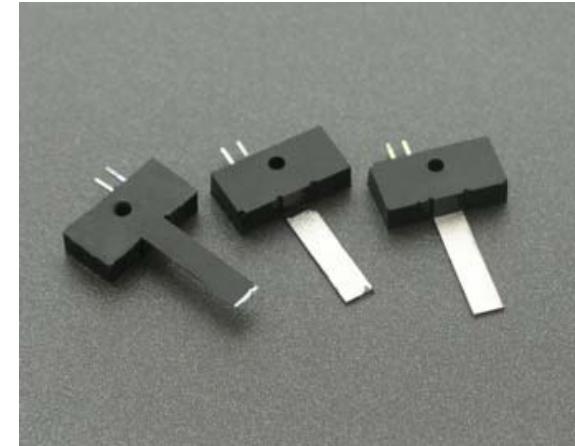
- Fabrication and microintegration on one step
- low-cost-perspective via insert inject molding
- extremely high design flexibility
- generators and actuators from one fabrication technology



# Examples of PPC microdevices



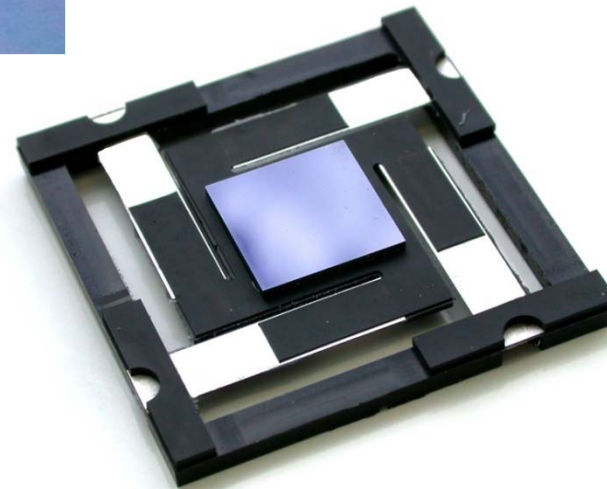
*nanojet dispenser*



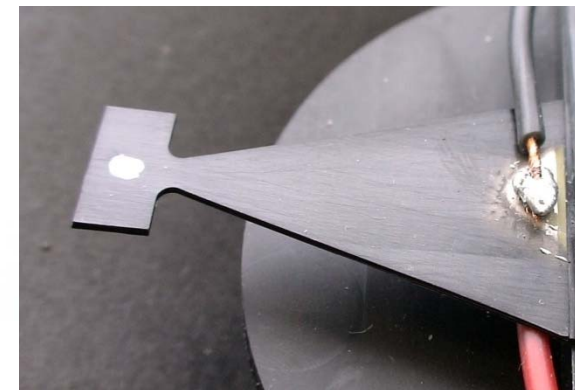
*various piezogenerators*



*microvalve*



*2D tilting mirror*





# Piezoelectric materials: constitutive equations

## Material parameters

$d$  : piezoelectric strain coefficient  $\left[ \frac{m}{V} \right]$

$e$  : dielectric constant  $\left[ \frac{As}{Vm} \right]$

$Y$  : Young's modulus  $\left[ \frac{N}{m^2} \right]$

$$\underbrace{\varepsilon}_{\text{Hooke's Law}} = \frac{\sigma}{Y} + \underbrace{d \cdot E}_{\text{piezoelectric coupling}}$$

## Variables

$\sigma$  : mechanical stress  $\left[ \frac{N}{m^2} \right]$

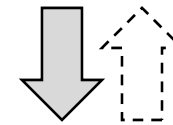
$\varepsilon$  : mechanical strain

$E$  : electrical field strength  $\left[ \frac{V}{m} \right]$

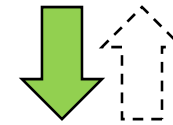
$D$  : electrical displacement  $\left[ \frac{As}{m^2} \right]$

$$\underbrace{D}_{\text{dielectric equation}} = e \cdot E + \underbrace{d \cdot \sigma}_{\text{piezoelectric coupling}}$$

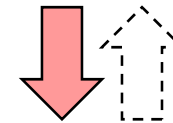
mechanical  
energy



mechanical  
deformation



charge  
displacement



electrical  
energy

# Typical piezoelectric material parameters



*One could think:  
„Soft is better“*

|                                         | units                      | „hard“<br>PZT<br>PIC 181 | „soft“<br>PZT<br>PIC 255 | biaxial<br>PVDF                      | PZN-PT<br>001P-<br>33(31)T |
|-----------------------------------------|----------------------------|--------------------------|--------------------------|--------------------------------------|----------------------------|
| strain coefficient $d_{31}$             | $10^{-12}$ As/N            | - 120                    | - 180                    | 6...10                               | - 1400                     |
| strain coefficient $d_{33}$             | $10^{-12}$ As/N            | + 265                    | + 400                    | 13...22                              | + 2400                     |
| relative permittivity $\epsilon_{r,11}$ | ---                        | 1500                     | 1650                     | 10...12                              | ---                        |
| relative permittivity $\epsilon_{r,33}$ | ---                        | 1250                     | 1750                     | 10...12                              | 6500                       |
| Young's modulus $Y_{33}$                | $10^{10}$ N/m <sup>2</sup> | 16.6                     | ---                      | 0.16...0.22                          | ---                        |
|                                         |                            | PI Ceramics,<br>Germany  | Piezotech,<br>France     | Microfine<br>Materials,<br>Singapore |                            |

# Piezoelectric energy harvesting: Coupling, quality factor, Figure of Merit



## Piezoelectric coupling coefficient $k$

... defines the ability of a **piezoelectric material** to convert mechanical into electrical energy, here:

$$k = \sqrt{\frac{\text{electrical energy stored}}{\text{mechanical energy applied}}}$$

$$k = d \cdot \sqrt{\frac{Y}{e}}$$

## Quality factor $Q$

... defines, for a **resonant harvester**, ist „aptitude for resonance“.  
 $Q$  is, as for any oscillator, inversely proportional to damping  $\delta$ :

$$Q \approx \frac{1}{\delta}$$

## Figure of Merit $F$

... defines the power output of a piezoelectric generator:

$$P_{out} \approx \underbrace{k^2 \cdot Q^2}_{\text{Figure of Merit } F}$$

$$F = k^2 \cdot Q^2$$

*F. Goldschmidtboeing et al., J. Micromech. Microeng. 21, 2011, 045006.*

# Example for nonlinearities: Young's modulus $Y$

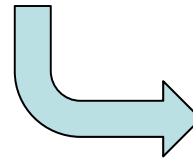
## Linear model (essentially Hooke's law)

$$\sigma(t) = Y_1 \cdot \varepsilon(t) + \dots$$

with

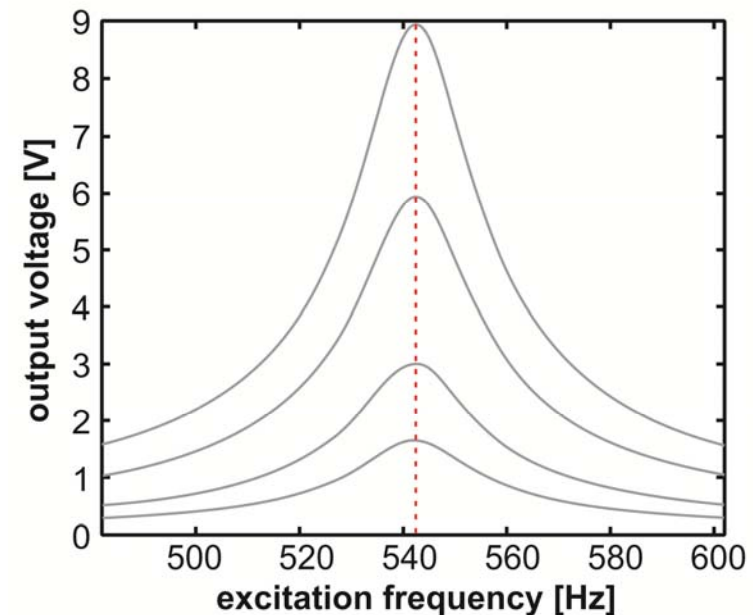
sinusoidal

excitation:  $\varepsilon(t) = \varepsilon_0 \cdot \sin(\omega t)$



## Modeling results

- **symmetric** behaviour at resonance
- **constant resonance frequency** at variable excitation amplitudes



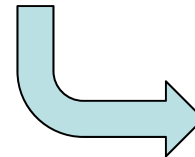
S. C. Stanton et al., *Journal of Intelligent Material Systems and Struct.* 23, 2012, 183.

# Example for nonlinearities: Young's modulus $Y$

## Cubic model

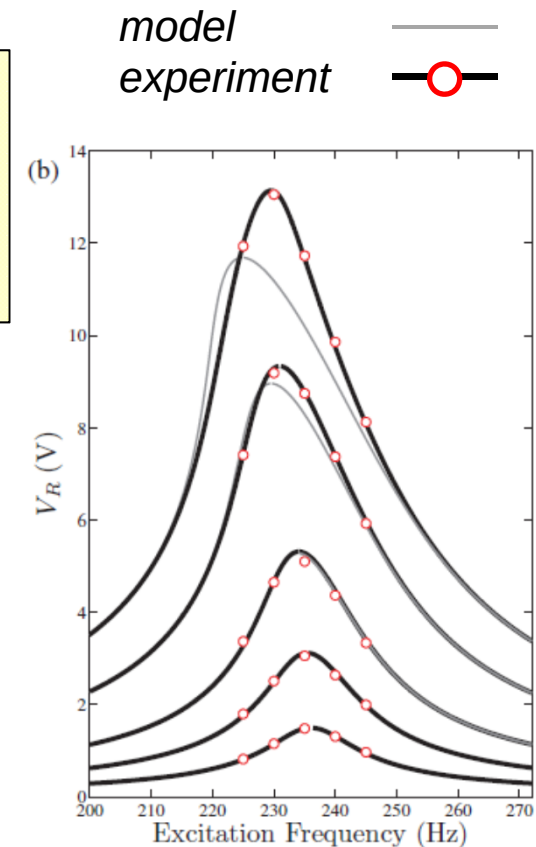
$$\sigma(t) = Y_1 \cdot \varepsilon(t) + \underbrace{\frac{1}{2} \cdot Y_2 \cdot \varepsilon(t)^2}_{\text{cancels out for bending}} + \frac{1}{6} \cdot Y_3 \cdot \varepsilon(t)^3 + \dots$$

with sinusoidal excitation:  $\varepsilon(t) = \varepsilon_0 \cdot \sin(\omega t)$   
 $\varepsilon(t)^3 = \varepsilon_0^3 \cdot \sin^3(\omega t)$



## Modeling results

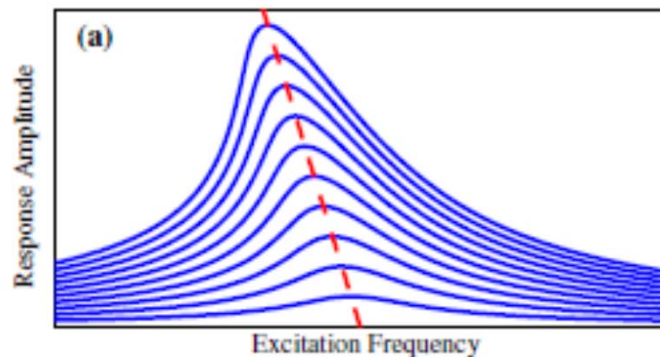
- Resonance curves **tilted to lower frequencies**
- Tilting increases severely with excitation amplitude  
**(in contrast to experimental observations)**



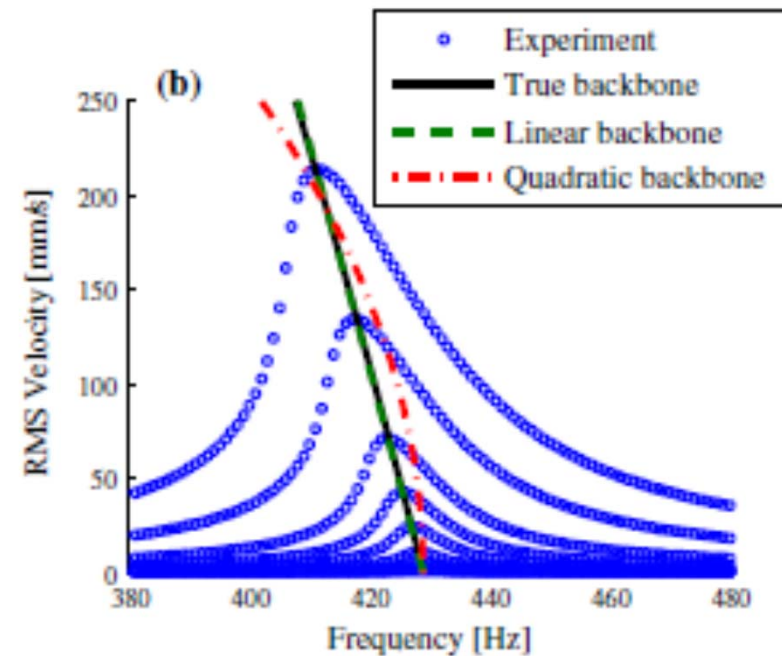
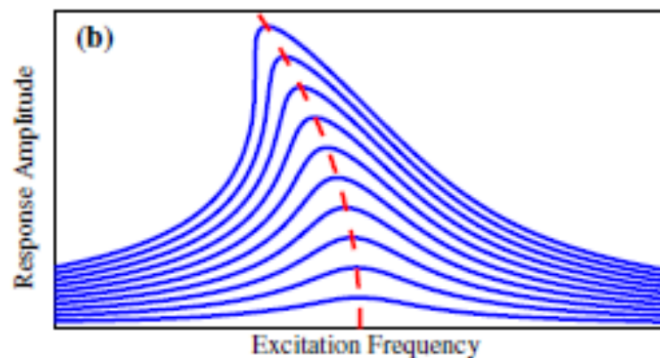
S. C. Stanton et al., *Journal of Intelligent Material Systems and Struct.* 23, 2012, 183.

# Example for nonlinearities: Young's modulus $Y$

*Experimental results*



*Results from a cubic model*



S. Leadenham, A. Erturk,  
*Nonlinear Dynamics* 79, 2015, 1727-1743.

# Example for nonlinearities: Young's modulus $Y$



## Adapted model with „amplitude nonlinearity“

$$\sigma(t) = Y_1 \cdot \varepsilon(t) + \frac{1}{2} \cdot Y_2 \cdot \varepsilon(t)^2 + \frac{1}{6} \cdot Y_3 \cdot \varepsilon(t)^3 + \dots$$

With usual „full nonlinearity“:

$$\varepsilon(t) = \varepsilon_0 \cdot \sin(\omega t)$$

$$\varepsilon(t)^2 = \varepsilon_0^2 \cdot \sin^2(\omega t)$$

$$\varepsilon(t)^3 = \varepsilon_0^3 \cdot \sin^3(\omega t)$$

Using „**amplitude nonlinearity**“ caused by the ferroelectric hysteresis and taking only the second order terms:

$$\varepsilon(t) = \varepsilon_0 \cdot \sin(\omega t)$$

$$\varepsilon(t)^2 \rightarrow \varepsilon_0 \cdot \varepsilon_0 \cdot \sin(\omega t) = \varepsilon_0^2 \cdot \sin(\omega t)$$

$$\sigma(t) = Y_1 \cdot \varepsilon(t) + \frac{1}{2} \cdot Y_2 \cdot \varepsilon_0 \cdot \varepsilon(t)$$



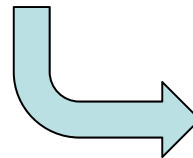
# Comparison to measurements: Adapted nonlinear model

$$\sigma(t) = Y_1 \cdot \varepsilon(t) + \frac{1}{2} \cdot Y_2 \cdot \varepsilon_0 \cdot \varepsilon(t)$$

With

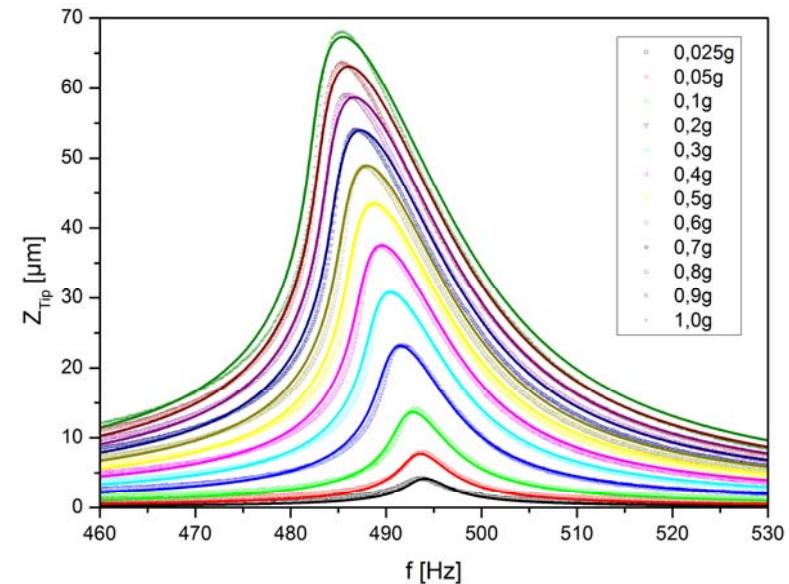
sinusoidal

excitation:  $\varepsilon(t) = \varepsilon_0 \cdot \sin(\omega t)$



## Modeling and experimental results

- good fit for small and large excitations
- good prediction of ...
  - resonance frequency shift
  - amplitude shift
- some consequences for material choice



F. Goldschmidtboing, S. Neiss, M. Kröner, P. Woias, *Proc. MST-Kongress 2013*, 532-535. (in German)

F. Goldschmidtboeing, C. Eichhorn, M. Wischke, M. Kroener, P. Woias, *Proc. PowerMEMS 2011*, 114-117.

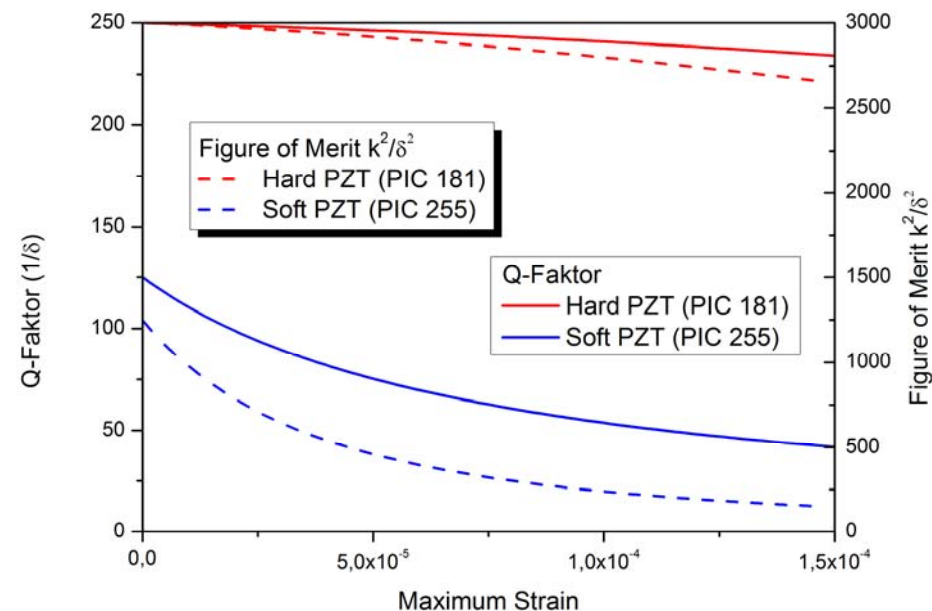
# Quality and power factor with the adapted nonlinear model

**„Hard“ vs. „soft“ PZT:  
Theoretical and  
experimental  
observations**

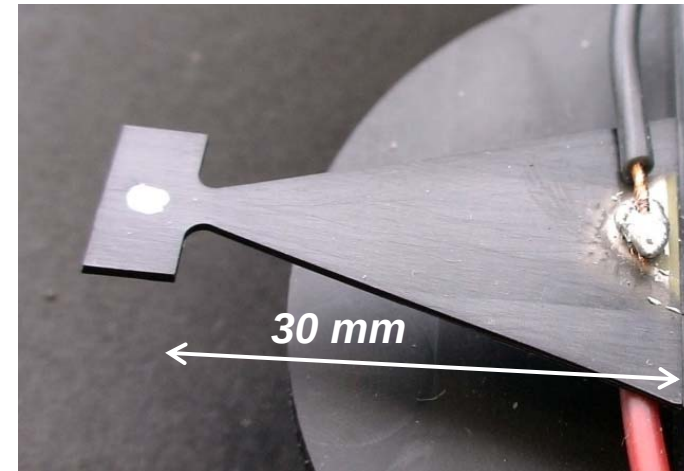
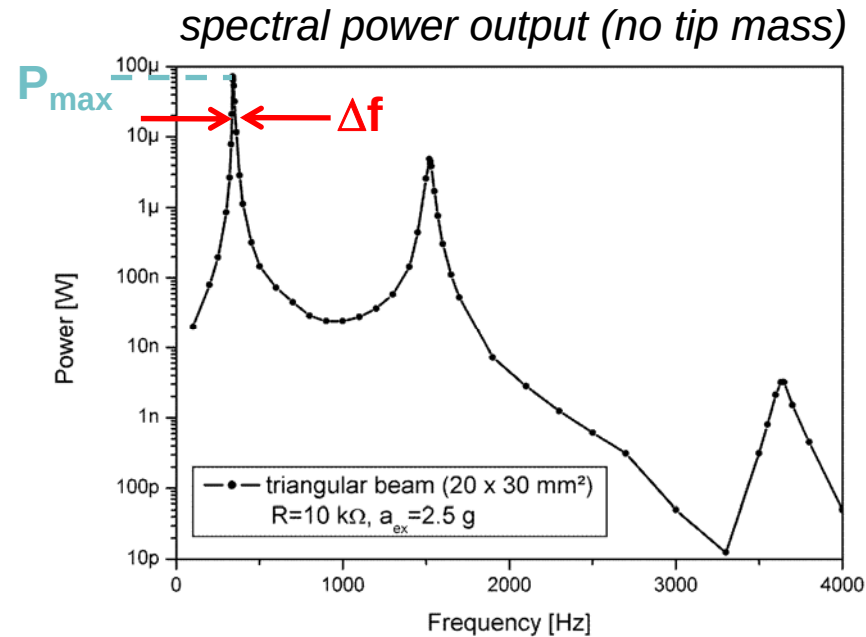
|                    |                        | „Hard“ | „Soft“ |
|--------------------|------------------------|--------|--------|
| Material property  | Charge coeff. $d_{31}$ |        | +      |
| Generator property | Quality factor Q       | +++    |        |
|                    | Figure of Merit F      | +++    |        |
|                    | Power bandwidth        | lower  | higher |

„Hard“ PZT shows a larger Q and F, hence a higher output power, despite its lower charge coefficient  $d_{31}$ .

F. Goldschmidtboïng, S. Neiss, M. Kröner, P. Woias, *Proc. MST-Kongress 2013*, 532-535. (in German)



# Typical (non-adaptive) power output



F. Goldschmidtböing, P. Woias,  
*JMM* 18, 2008, 104013

## Characteristics of resonant operation

- highest output power  $P_{\max}$  at resonance only
- small power bandwidth  $\Delta f$  at high Q factor and conversion efficiency
- **not adapted to variable input frequencies**

# Frequency-tunable piezo generators ?

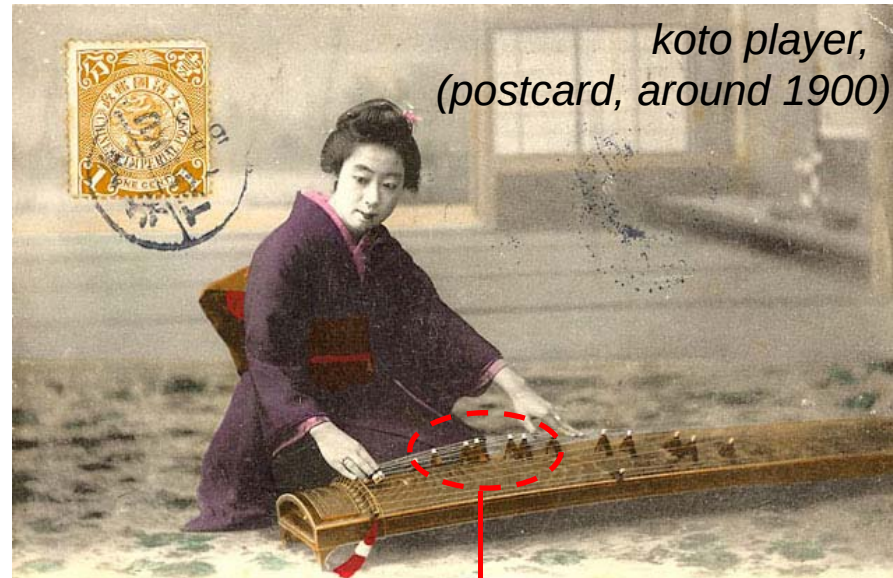
## Principle and inspiration

- continuous re-tuning of a mechanical oscillator
- similar to the tuning of a musical string instrument



robot violinist

© Toyota, 2010

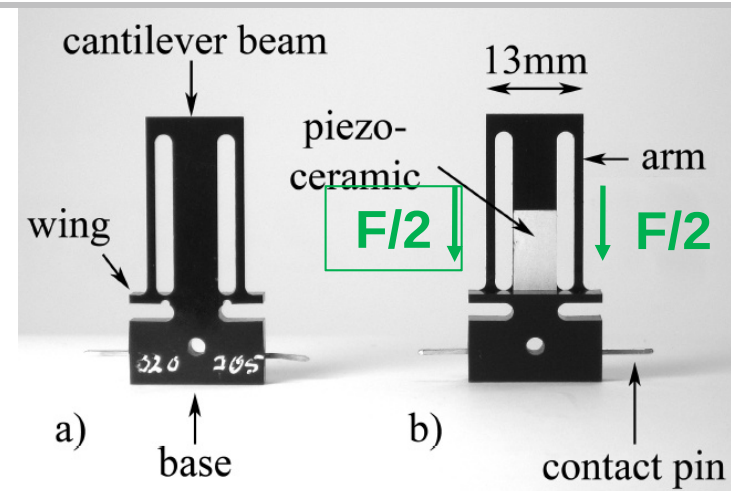
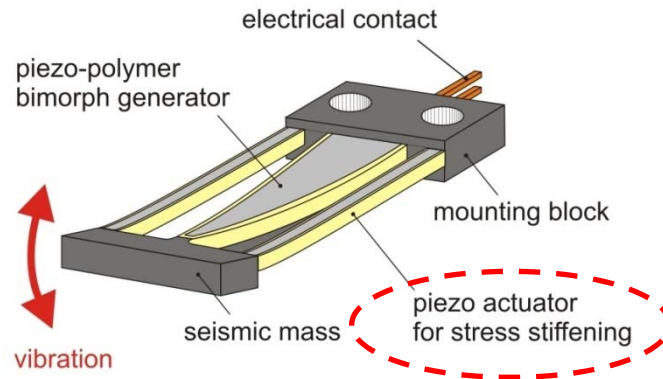


koto player,  
(postcard, around 1900)

## Frequency tuning via change of ...

- mechanical stress
- size (length)
- seismic mass

# Frequency-tunable piezo generator

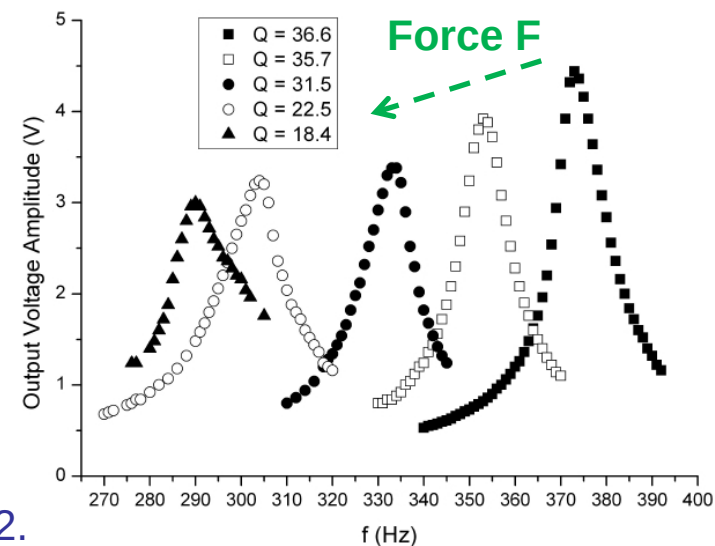


## Principle of „stress tuning“

- Actuation force in the „arms“ will stiffen the resonating beam and thus change its resonance frequency

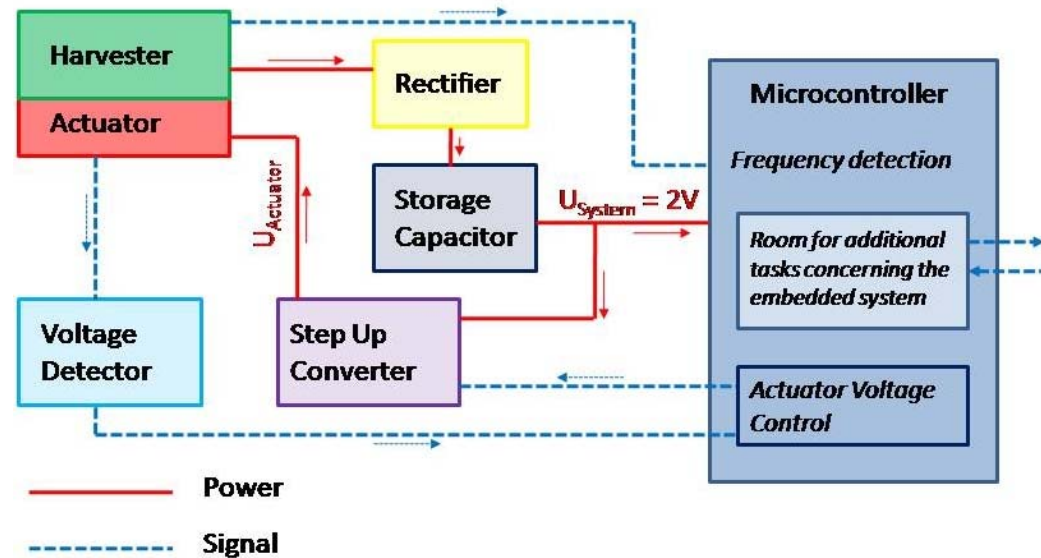
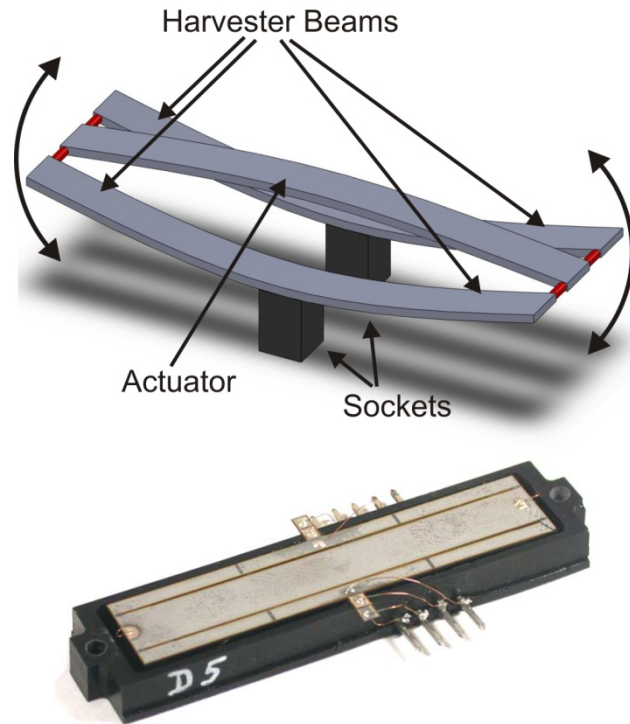
➔ **high tuning range (22%)**

➔ **loss of Q factor with increasing force**



C. Eichhorn et al., *Proc. PowerMEMS 2008*, 309-312.





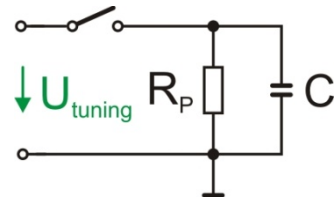
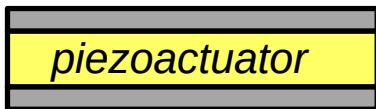
C. Eichhorn et al., Proc.  
PowerMEMS 2010, 207-210.

## Fundamental questions

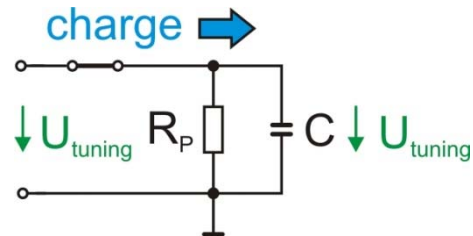
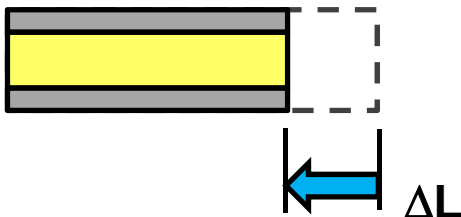
- How often will a „re-tuning“ be required ?
- Will the tuning operation itself „eat“ all the harvested power ?
- If so, how to avoid this ?

# Quasi-static tuning of piezoactuators

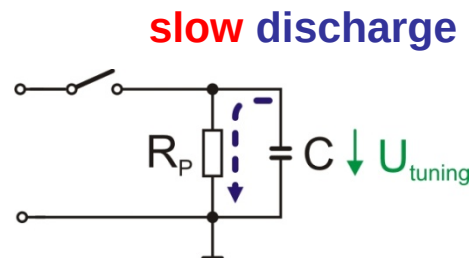
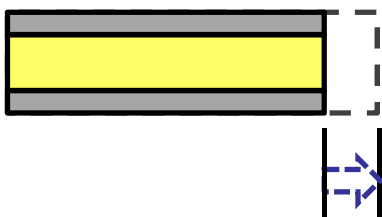
## 1. untuned



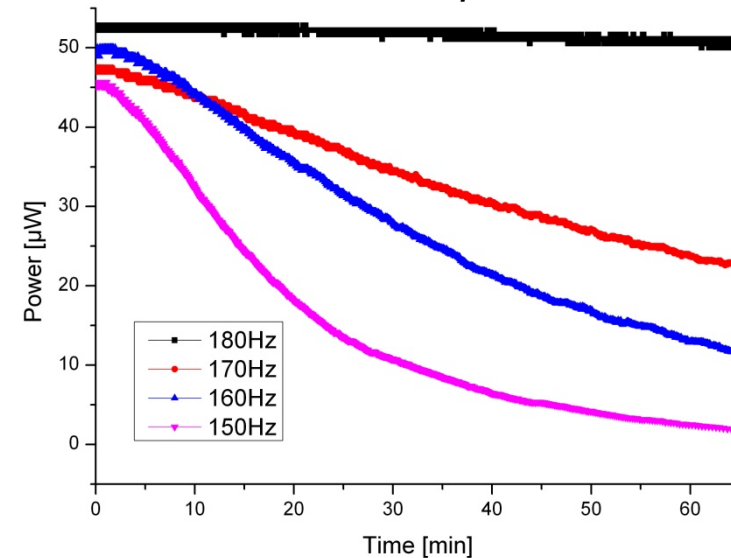
## 2. fast tuning



## 3. slow relaxation



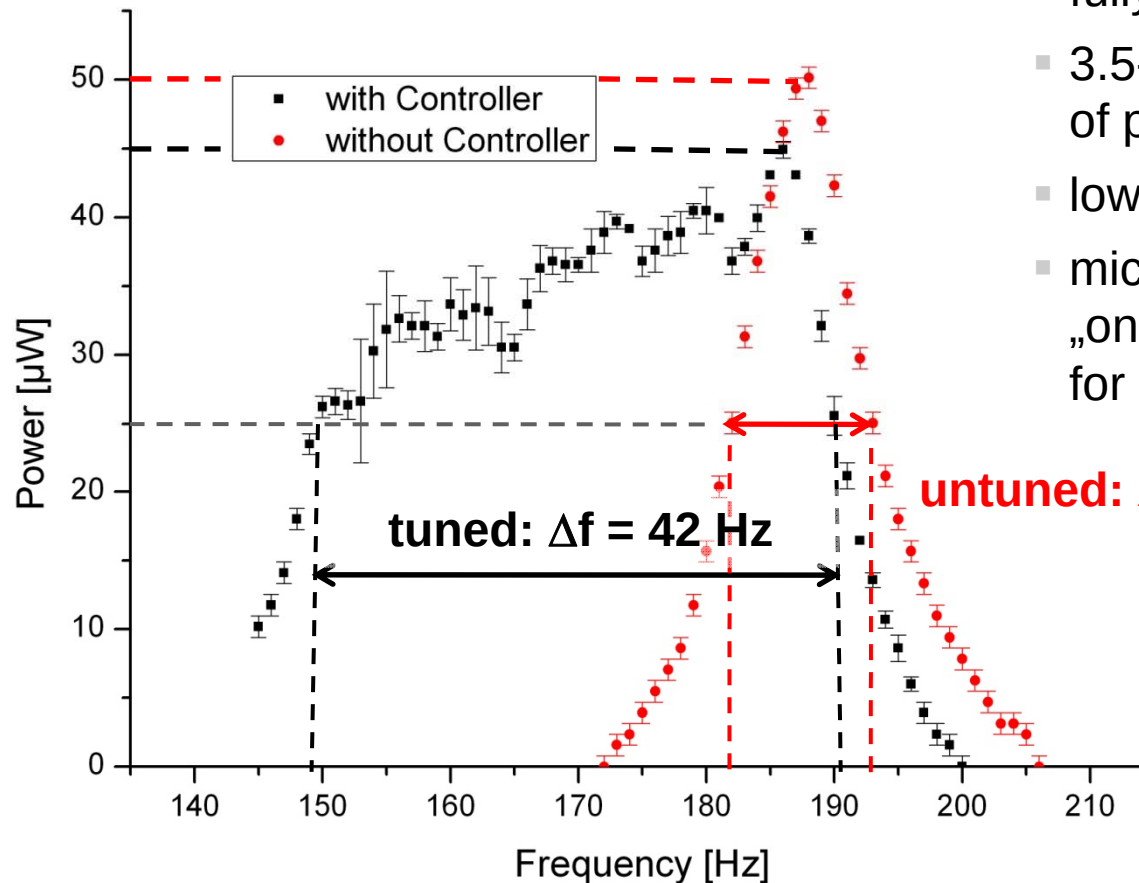
*effect of relaxation on the harvested power vs. time*



## Advantages

- piezoactuator stores charge and position
- only slow relaxation due to leakage currents

# System characteristics



- fully self-powered system
- 3.5-fold increase of power bandwidth  $\Delta f$
- low tuning power (5...25  $\mu\text{W}$ )
- microcontroller already „on board“, can be used for other purposes as well

untuned:  $\Delta f = 12 \text{ Hz}$

tuned:  $\Delta f = 42 \text{ Hz}$

output power in tuned and un-tuned operation  
(tuning intervals: 20 s,  
acceleration: 0.6 G)

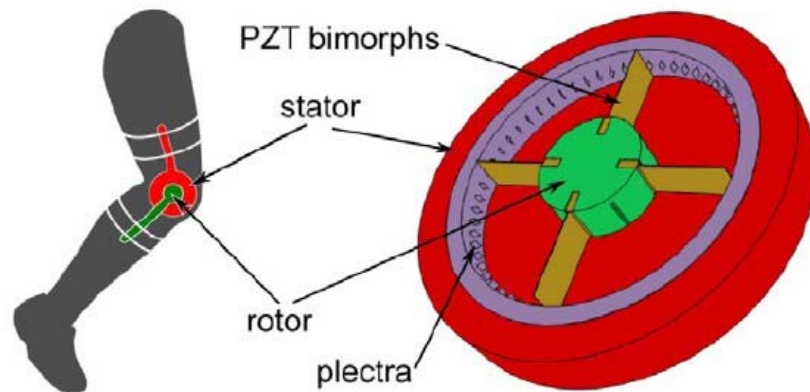
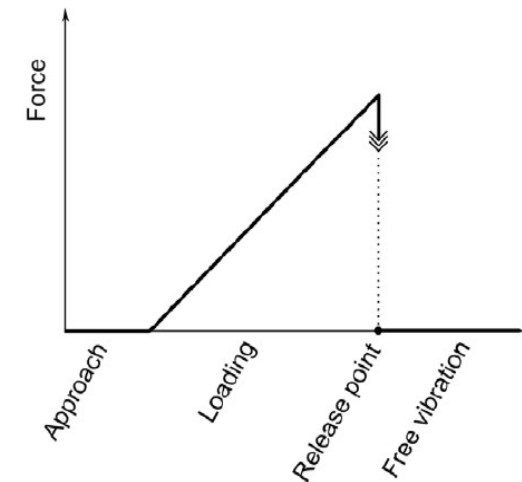
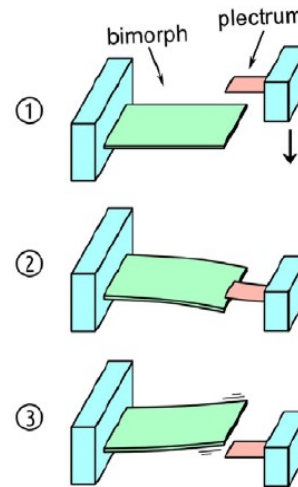
C. Eichhorn et al., *Proc. PowerMEMS 2010*, 207-210.



# Plucking-type piezogenerators

## Example: Knee-joint harvester

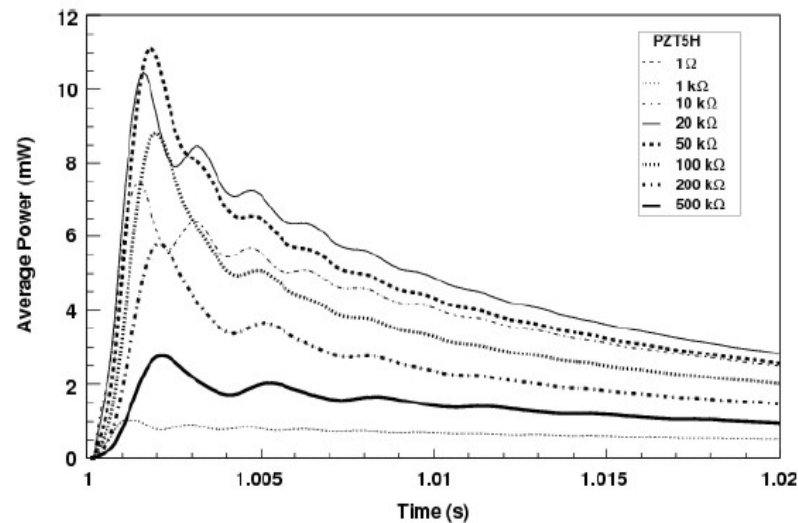
- central hub with piezo bimorphs
- plucking via a ring of plectrums during angular movements of the knee joint



M. Pozzi, M. Zhu,  
*Smart Mater. Struct.* 20, 2011, 055007.

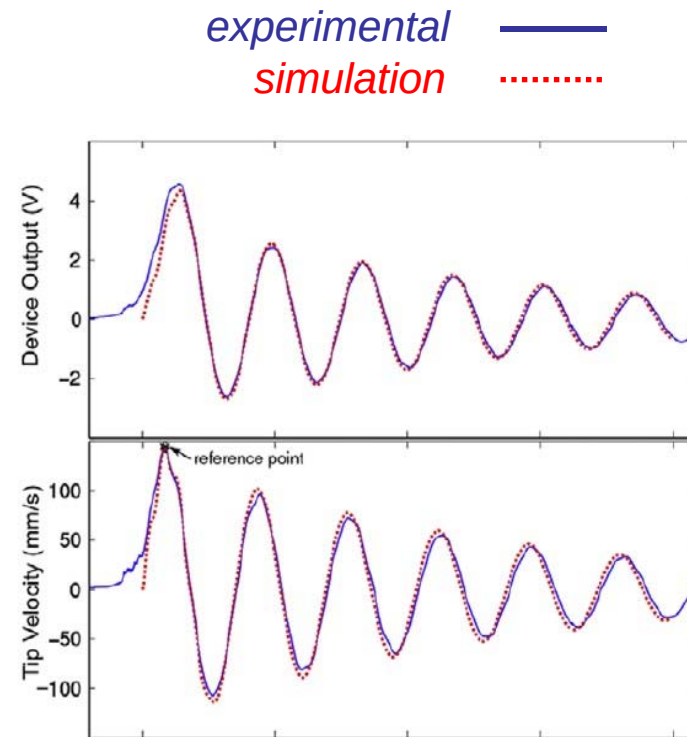
# Plucking-type piezogenerators

## Knee-joint harvester: output voltage and output power



*calculated output power during one plucking event (averaged values)*

M. Pozzi, M. Zhu,  
*Smart Mater. Struct.* 20, 2011, 055007

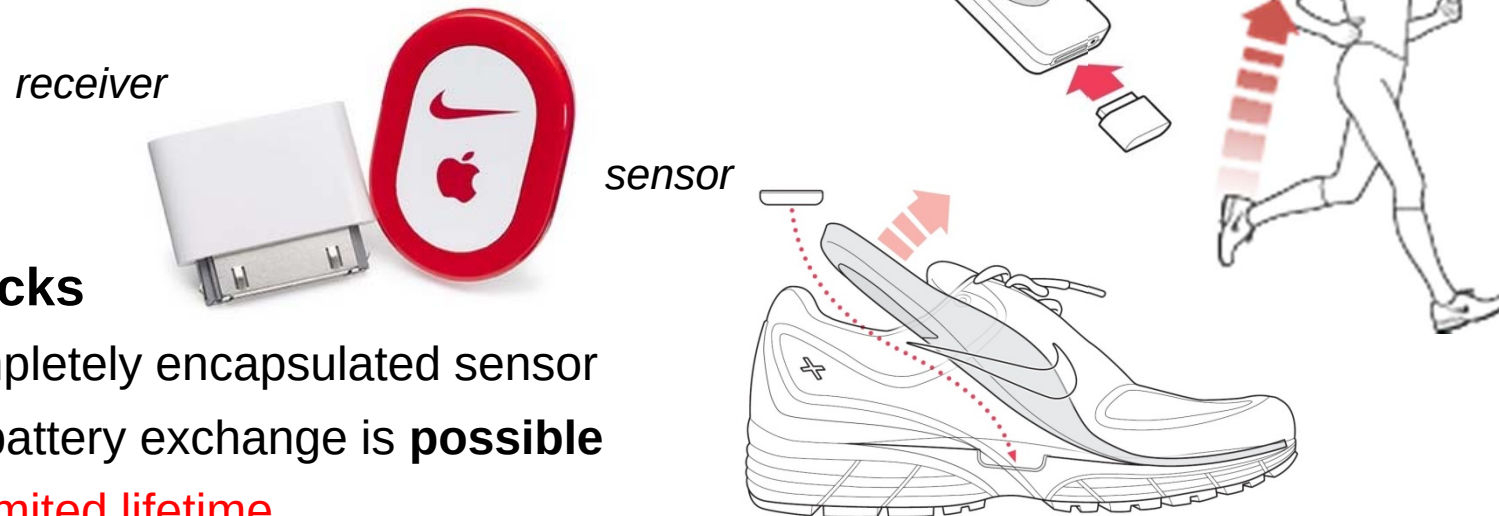


*simulated and measured  
tip displacement and output voltage*

# Energy-autonomous pedometers

## Battery-powered pedometer: example Nike+

- battery-powered pedometer and wireless transmitter in the sole of a sports shoe
- wireless receiver at the Apple iPod™



## Drawbacks

- completely encapsulated sensor
- no battery exchange is **possible**
  - ➔ limited lifetime
  - ➔ sensor with empty battery = waste
  - ➔ energy-autonomous alternative ?

© Apple, Nike

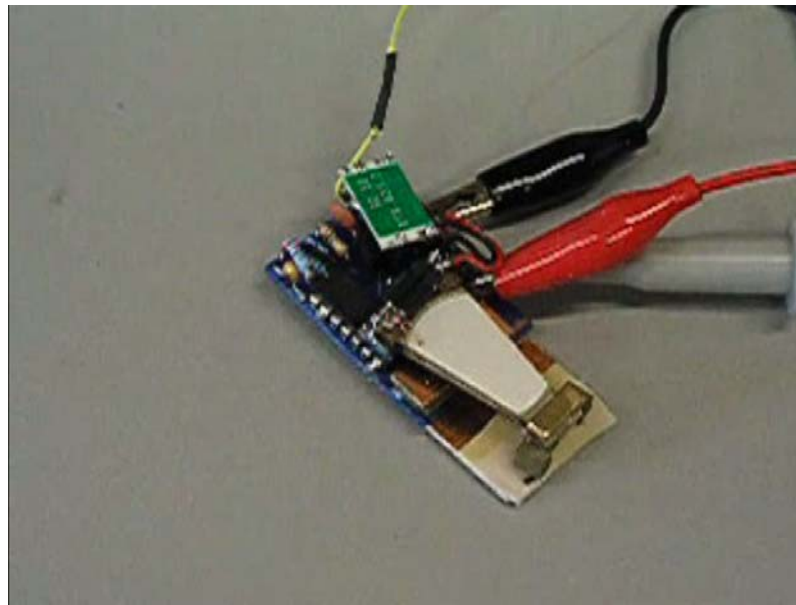
# Energy-autonomous pedometer

## Concept

- impact-type piezogenerator in a small „add-on package“ mounted at a sports shoe

*typical output power: 80  $\mu$ W*

- pedometer and wireless transmitter at/in the shoe
- wireless receiver in a sports watch





# Autonomous sensors in tunnel buildings



- environmental monitoring (temperature, humidity, ventilation, ...)
- detection of fire, explosions, earthquake, ...
- monitoring of the building's structural health



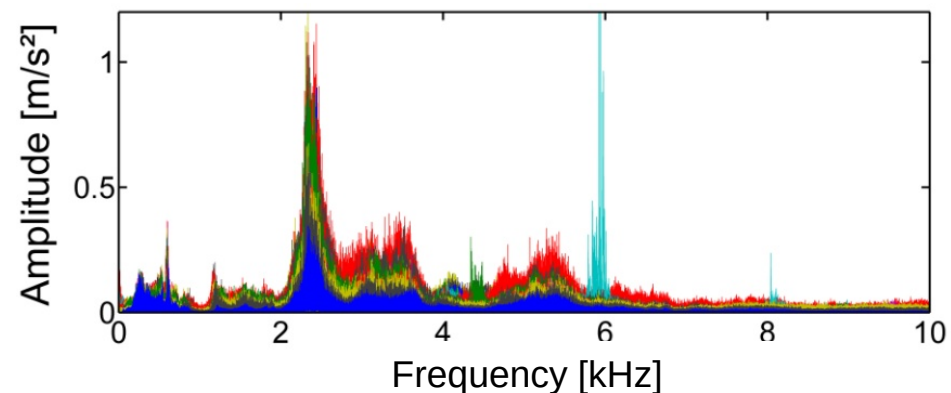
# Harvesting from vibrations at the rail

## Application: train monitoring

- detection of train passage
- measurement of train velocity
- detection of train stops

## But first: measurement of the available vibrations

- wide frequency spectrum
- different levels at different locations
- severe influence of train (passenger train, freight train)

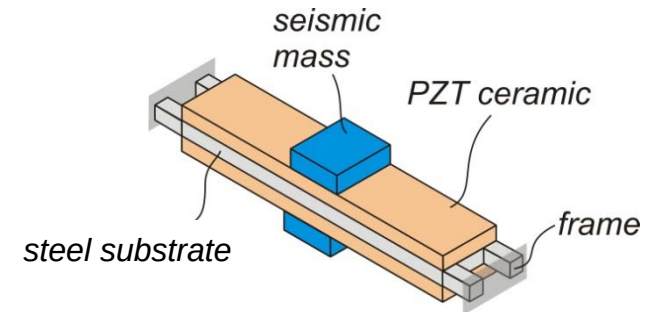


*vertical acceleration spectrum from 57 trains at a modern concret rail sleeper, measured in the Arlberg tunnel, Austria*

# Harvesting from vibrations at the rail

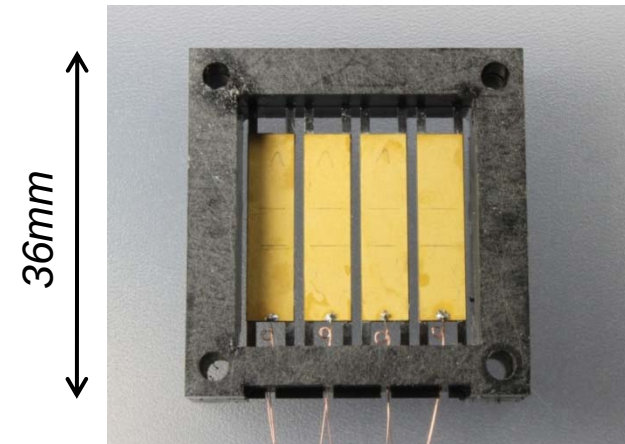
## Generator and power management

- array of four piezogenerators with different resonance frequencies
- low power start-up electronics
- on-site test under real conditions (Loetschberg basis tunnel, Switzerland)



*rail sleeper*

*piezogenerator array  
with power electronics*



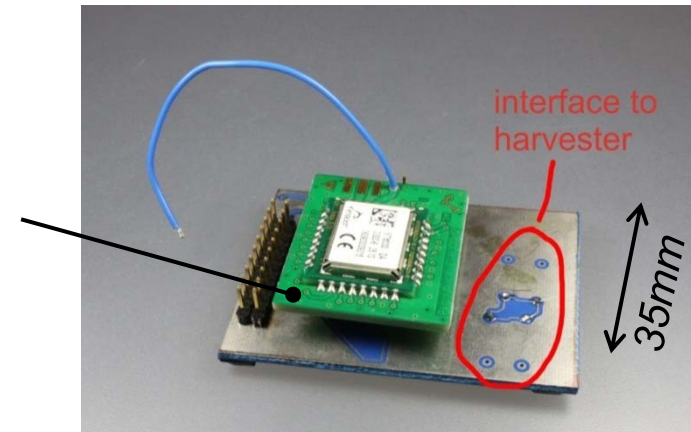
*piezogenerator array  
(without seismic masses)*



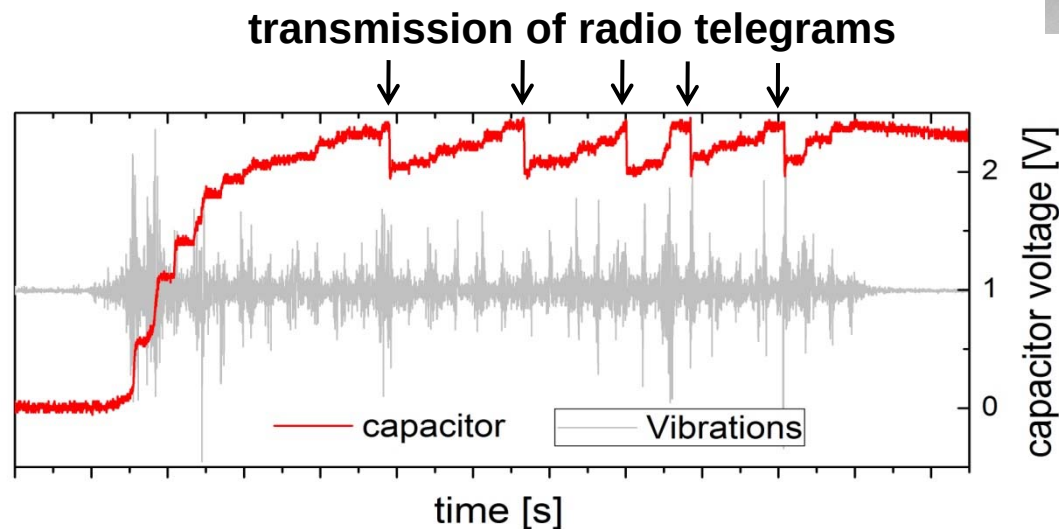
# Harvesting from vibrations at the rail

## System integration and test

- integration of harvester, storage capacitor, power management and wireless module (EnOcean STM 300)
- test on a shaker with real-life vibration pattern



*energy-autonomous  
train passage detector*



*system operation during the  
simulated passage of a freight  
train in the Loetschberg basis  
tunnel*

# Summary and conclusions

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The design of a piezoelectric generator is a complex and “multidimensional” approach, taking into account the ...

- nature of the mechanical energy to be harvested from,
- the design of appropriate piezo generators
- the appropriate choice of piezoelectric materials
- the establishment of system concepts, including active generator control,
- the development of suitable fabrication processes, and
- many application-specific requirements.

However, if done well, we are capable **today** to provide generators for ...

- high-efficiency mono-resonant energy harvesting,
- broadband energy harvesting,
- frequency-adapted harvesting systems, or
- harvesting from stochastic or regular low-frequency signals.

# Thank you very much for your attention !



## Up-coming one-day seminar on Energy Harvesting:

„Energy Harvesting“, late autumn 2016, Zuerich, Switzerland

details on: [www.fsrm.ch](http://www.fsrm.ch)

